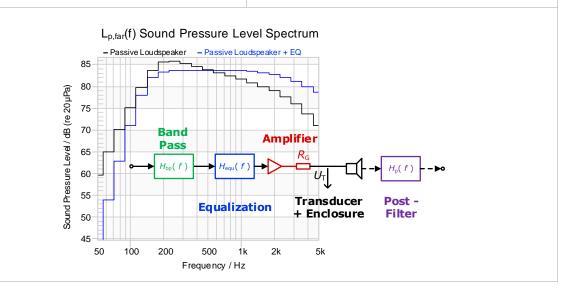
Module of the KLIPPEL ANALYZER SYSTEM (Document Revision 1.6)

FEATURES

- Linear signal modeling from digital input to acoustical output.
- Lumped network parameters for passive components
- Automatic equalization (DSP)
- Small signal performance for any audio input (music, test signal)
- Efficiency and voltage sensitivity versus frequency and broadband signals

BENEFITS

- Small signal performance in target application
- Considers digital, electrical, mechanical, acoustical components
- Minimum set of essential parameters
- Fast calculation of frequency responses
- Filter parameters for optimal system alignment
- Basis for large signal modeling (SIM)



DESCRIPTION

The LSIM Linear Simulation describes an active loudspeaker or headphone driver by using a linear lumped parameter model. Main components are equalizer, amplifier, transducer and enclosure. Using any selected input spectrum (e.g. music), meaningful statistical single values (e.g. mean efficiency) and various state spectra (e.g. SPL) are calculated. This is a useful base for defining transducer and amplifier requirements and providing significant information about the audio performance. Various transfer functions reveal the relationship between digital, electrical, mechanical and acoustical signals.

The LSIM features an easy-to-use simulation software with lumped or geometrical input parameters for initial (small signal) design, which is the basis for the large signal simulation in other Klippel software modules (SIM Simulation, SIM-AUR Auralization).

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CONTENT

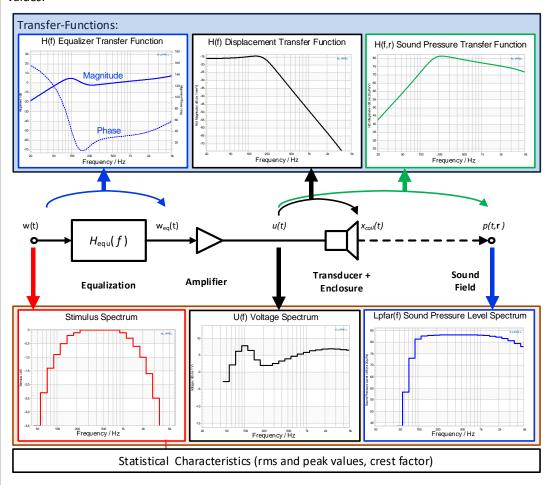
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1 Overview

1.1 Principle

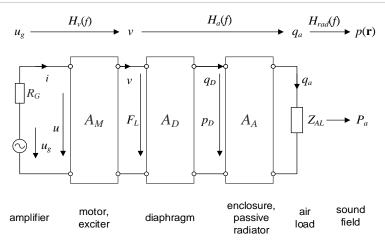
Basic Principle The LSIM Linear Simulation module illustrates a simplified linear active loudspeaker containing a band pass filter section for simulating a crossover, a prefilter (Equalization) specified by the transfer function $H_{\rm equ}(f)$, an amplifier with an output resistance of $R_{\rm g}$ and an electrodynamical transducer mounted in an enclosure. The optimal equalizer transfer function $H_{\rm equ}(f)$ for system alignment will be calculated automatically for a specified target transfer behaviour.

Signal based system design is possible by defining a relative input spectrum $G_w(f)$. Pink noise, typical program material according to *IEC 60268-21* and an option for individual external stimulus are provided. All spectra are converted into third octave spaced spectra. Based on this, state variables like U_g (amplifier output voltage without load) or U_T (terminal voltage) and further characteristics like $SPL_{\rm max}$ can be predicted. Entering a crest factor provides the option to estimate peak values.



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Lumped Parameter Model



The LSIM Linear Simulation module uses a lumped-parameter model of an electro-dynamical transducer mounted in common enclosures. This model is based on chain matrices describing the different parts of the loudspeaker. A_M describes the motor and mechanical behavior of the exciter, the diaphragm A_D , the enclosure A_A and passive acoustical elements like port or passive radiator. Employing this knowledge, total sound pressure level SPL(f), state variables (e.g. V_c), transfer functions such as $H_X(f)$ or the electrical impedance $Z_{\rm el}(f)$, as well as efficiency $\eta(f)$ and voltage sensitivity can be easily simulated.

Note that the *LSIM* module only simulates the linear behavior of the system, which is considered valid at small amplitudes. Please see *SIM Simulation* or *SIM-AUR Simulation / Auralization* for nonlinear modeling.

1.2 Input

Input Parameters

The LSIM input is structured into 4 categories:

Transducer:

• Linear transducer parameters (free air)

Enclosure:

- Type
- Geometrical properties or lumped parameters

Equalization:

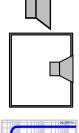
- High pass filter alignment
- User defined transfer behavior

Stimulus:

- Pink noise
- Typical program material according to IEC 60268-21

 S_d

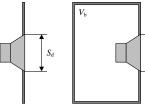
• User defined spectrum (e.g. music)

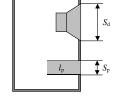


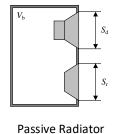




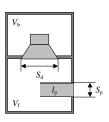
The LSIM supports the following enclosure types:







Box

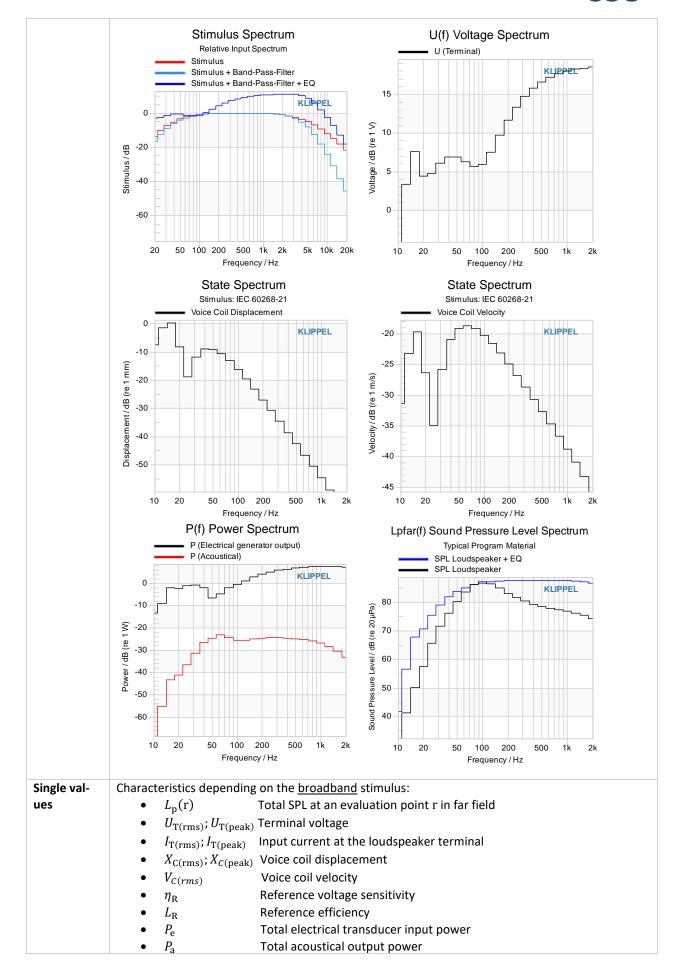


Baffle Closed Box

Vented Box

Bandpass System

Linear Transfer	The magnitude and phase frequency responses are calculated between the following state variables
Functions	• Sound pressure level $L_{\rm p}(f, r)$ in far field
	Displacement (voice coil, passive radiator)
	 Velocities (voice coil, passive radiator)
	Forces in the mechanical system
	 Volume velocities in the acoustical system
	in relation to the terminal voltage $U_{ m T}$. The electrical input impedance $Z_{ m e}(f)$ is also presented.
Reference	• Voltage sensitivity $L(f, \mathbf{r})$ versus frequency of a sinusoidal stimulus referenced to $u_{\mathrm{ref}} =$
Sensitivity	1 V and $r_{\rm ref}=1{ m m}.$
	• Reference voltage sensitivity $L_{\rm r}$ for the given broadband stimulus in accordance to IEC 60268-22.
Efficiency	• Efficiency $\eta(f)$ versus frequency of a sinusoidal stimulus.
	• Reference efficiency η_r for given broadband stimulus in accordance to IEC 60268-22.
Spectra	For a given broadband stimulus spectrum, the following 1/3 rd octave spectra are available:
based on	 Stimulus (with no filter, band-pass filtered, or band-pass and equalized)
Stimulus	 Internal state variables (e.g. Displacement for given stimulus)
	Power (electrical input, acoustical output)



2 Example

2.1 Simulation of a closed box system

Targets

The target of this example is to show a typical workflow on how to use the *LSIM* for active loudspeaker design. The task is to use the *LSIM* for designing a two-way closed box loudspeaker. We define the following design targets:

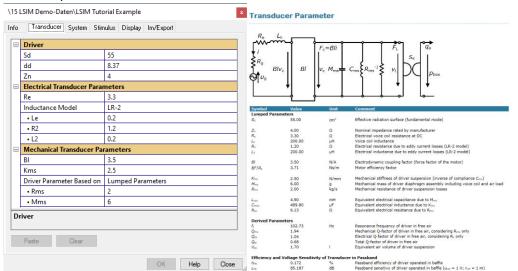
- The desired SPL output is 95 dB in 1 m distance in half-room.
- The volume of the box is limited to 1 I due to design choices.
- A 4" transducer with a maximum specified displacement of $x_{\text{max}} = 5 \text{ mm}$ shall be used.
- Crossover frequency to the tweeter f_{xo} =5 kHz

The following critical single values should be determined for typical program material with respect to these design targets:

- Power and voltage consumption (required for selecting a fitting amplifier)
- Efficiency and Voltage Sensitivity
- Peak voice coil displacement

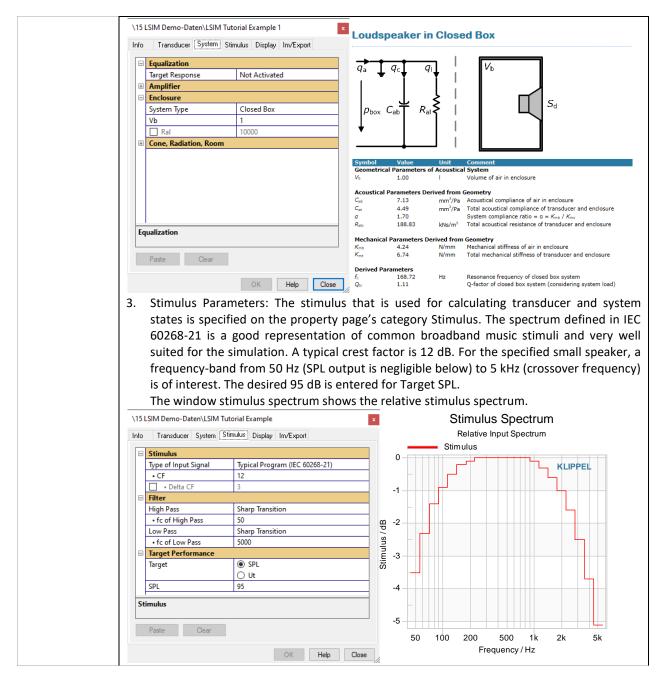
Parameters

1. Linear Transducer Parameters: Data of a small midrange speaker was imported from an LPM operation.



2. System parameters: The closed box loudspeaker is specified with a volume of 1 l. In the first step no equalization is applied to the system.

For simplification, no amplifier output resistance Rg, leakage losses Ral and post-filter is selected. The window Table System Parameters shows the headline Loudspeaker in Closed Box indicating the selected enclosure system. The picture below shows the corresponding equivalent circuit. The table below contains all derived enclosure parameters based on the entered closed-box parameters.



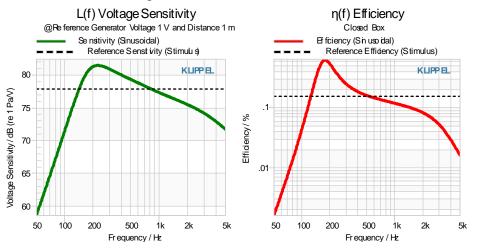
Results

The single values listed in table *State Variables* provide the most important information considering the simulated music reproduction:

- 1. For generating 95 dB SPL output using the desired IEC signal, the amplifier has to provide about 11 W. The peak voltage for an estimated crest factor of 12 is approx. 28 V.
- 2. The resulting effective (*reference*) efficiency is approx. 0.16 % for the selected stimulus. The effective (*reference*) voltage sensitivity is 78 dB.
- 3. For the specified stimulus, a peak displacement of approx. 2.86 mm is expected.

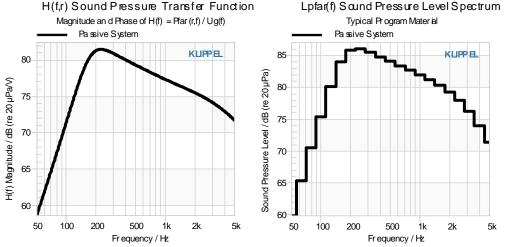
These values are the basis for defining the amplifier and transducer requirements. Checking the limits defined in the task above reveals that the desired SPL is achievable without exceeding the transducer's maximum displacement.

Viewing the curves *efficiency and voltage sensitivity versus frequency* is useful to understand the limitations of the passive loudspeaker system. The efficiency at lower frequencies decreases rapidly, so pushing frequencies to very low frequencies will be inefficient. Pay attention: Efficiency and voltage-sensitivity are not equal. Efficiency shows the ratio between incoming and outgoing power in percent. Voltage sensitivity shows the SPL output at 1 m distance with a terminal voltage of 1 V.



Adding Equalization

As the sound pressure transfer function (see chart below) shows, the frequency response is not equalized and the cutoff frequency is very high ($^{\sim}170~\text{Hz}$).



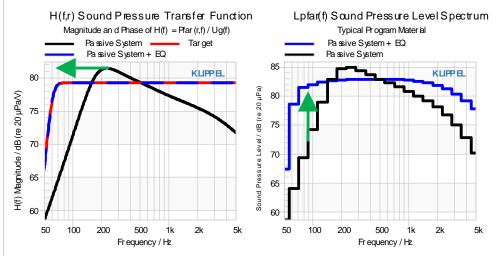
This section explains how to reduce the active system's cut-off frequency and how to apply equalization at higher frequencies.

Finding a good compromise for the optimal target high-pass characteristic is crucial for optimum performance: Setting the cut-off frequency too low will result in an inefficient system prone to extreme voltage requirements. Setting the cut-off frequency too high leads to a lack of bass and wasted potential. For the first simulation we define the following target response:

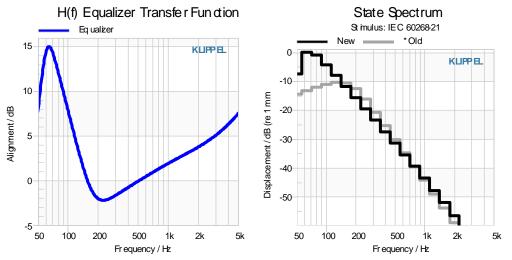
- Cut-off frequency: 60 Hz
- Filter type: 6th order Chebyshev filter

A 6th order filter is used to avoid wasting voltage and displacement below the cut-off frequen-

cy where the voltage sensitivity drops significantly (Compare H(f,r) Sound Pressure and H(f) Displacement). See the sound pressure transfer function and spectrum which shows the impact of the alignment filter on the sound pressure.



Applying this alignment filter results in a peak voltage of approx. 47 V and an electrical power of almost 34 W to achieve the desired 95 dB SPL output. The required peak displacement is 6.6 mm now. These values have increased dramatically compared to the passive system. This is caused by the excessive bass boost defined by the target alignment. As can be seen in the chart below, the alignment is boosting low frequencies up to 15 dB. In the *State Variables* window, the effective voltage boost L_{EQ} for the selected stimulus is displayed (in this case the boost is 5.5 dB).



As the estimated displacement exceeds the transducer specification and the electrical power will heat up the voice coil too much (typical thermal resistance of this transducer type is 3...5 K/W) reducing efficiency, the initially used cut-off frequency should be increased.

For the next simulation we use the following target alignment:

Cut-off frequency: 75 Hz

- Filter type: 6th order Chebyshev filter

Now the peak voice coil displacement is below the limit ($x_{c,peak} = 4.7$ mm), the electrical power for this stimulus is 20 W and the peak voltage is 38 V. If the amplifier can deliver this voltage, the chosen target alignment is reasonable. If a weaker amplifier is used, the cut-off frequency has to be increased or the design targets to be relaxed.

Comparing the reference efficiency and reference voltage sensitivity between the passive system and the equalized system reveals that the bass enhancement results in a speaker with linear frequency response and much better low frequency response at the cost of lower efficiency:

- $L_{R, \text{ old}} = 77.9 \text{ dB}$; $\eta_{R, \text{ old}} = 0.16 \%$;

- $L_{R, \text{ new}} = 75.4 \text{ dB}; \eta_{R, \text{ new}} = 0.09 \text{ %};$

3 Requirements

3.1 Hardware	
License Device Klippel Dongle or Klippel Analyzer 3 may be used to run this product.	
3.2 Software	
dB-Lab (>210.560)	dB-Lab is the project management software of the KLIPPEL R&D SYSTEM.

4 Parameter

4.1 Input

4.1.1 Electro Dynamic Transducer

Parameter	Symbol	Unit
Effective radiation surface	$S_{ m d}$	cm ²
Diameter of round effective radiation surface	$d_{ m d}$	cm
Nominal impedance rated by manufacturer	$Z_{ m n}$	Ω
Electrical voice-coil resistance at DC	R_{e}	Ω
Voice coil inductance	$L_{ m e}$	mH
Electric resistance due to eddy current losses	R_2	Ω
Electrical inductance due to eddy current losses	L_2	mH
Electric resistance due to eddy current losses	R_3	Ω
Electrical inductance due to eddy current losses	L_3	mH
Factor in LEACH model	K	Ω
Exponent in LEACH model	n	
Factor of real part in WRIGHT model	$K_{ m rm}$	Ω
Exponent of real part in WRIGHT model	$E_{ m rm}$	
Factor of imaginary part in WRIGHT model	K_{xm}	Ω
Exponent of imaginary part in WRIGHT model	$E_{ m xm}$	
Effective instantaneous electrodynamic coupling factor (force factor of the mo-	****	
tor) defined by the integral of the magnetic flux density B over the voice coil	Bl	N/A
length I Mechanical stiffness of driver suspension (inverse of compliance C _{ms})	K _{ms}	N/mm
Mechanical resistance of driver suspension losses	$R_{\rm ms}$	kg/s
Mechanical mass of driver diaphragm assembly including voice coil and air load	$M_{\rm ms}$	g g
Transducer resonance frequency (influences Rms and Mms)	$f_{\rm S}$	Hz
Mechanical Q-factor of driver in free air, considering Rms only (influences Rms)	Q_{ts}	

4.1.2 Equalization

High pass filter alignment:

Alignment Type:

- 1. Biquad filter
- 2. Bessel filter (4th and 6th order)
- 3. Chebyshev filter (4th and 6th order)
- 4. Butterworth filter (4th and 6th order)

Parameter	Symbol	Unit
Target Cutoff Frequency	f_0	Hz

4 Parameter

Chebyshev Constant	$C_{ m Chebyshev}$	
Arbitrary target transfer behavior		
Target response as matrix containing frequencies and corresponding levels		

4.1.3	Amplifier
7.1.3	Ampinici

Parameter	Symbol	Unit
Output-resistance of amplifier output including cables	R_{g}	Ω

4.1.4 Stimulus

Type of input signal:

- 1. Pink noise
- 2. Typical program (IEC 60268-21)
- 3. External spectrum

Bandpass:

- 1. Ideal (rectangle)
- 2. Butterworth

Parameter	Symbol	Unit
Cutoff frequency of the high pass filter	$f_{ m cHP}$	Hz
Slope of high pass filter	$m_{ m HP}$	dB
Cutoff frequency of the Low pass filter	$f_{ m cLP}$	Hz
Slope of low pass filter	$m_{ m LP}$	dB
Crest factor	CF	dB
Difference between crest factor for voltage and current signal and crest factor for displacement signal	ΔCF	dB

4.1.5 **Enclosure**

Enclosure type:

- 1. Baffle
- 2. Closed box
- Vented box (with slit or tube-shaped vent)
 Box with passive radiator
- 5. Bandpass system (with slit or tube-shaped vent)

Parameter	Symbol	Unit
Geometrical parameters:		
Volume of air in enclosure	$V_{ m b}$	1
Surface area of port	$S_{ m p}$	cm ²
Diameter of port	$d_{ m p}$	cm
Length of port	$l_{\rm p}$	cm
Width of surface area of port	$w_{\rm p}$	cm
Height of surface area of port	h_{p}	cm
Effective projected surface area of passive radiator diaphragm	$S_{ m r}$	cm ²
Diameter of round effective projected surface area of passive radiator dia-	d	cm
phragm	$d_{ m r}$	CIII
Volume of air in front enclosure	$V_{ m f}$	1
<u>Lumped parameters:</u>		
Acoustic resistance of losses due to leakage	$R_{\rm al}$	kNs/m ⁵
Acoustic mass of port including air load	$R_{\rm ap}$	kNs/m³
Acoustic resistance of port losses	$M_{\rm ap}$	kg/m ⁴
Mechanical mass of passive radiator diaphragm including voice coil and air load	$M_{ m mr}$	g
Mechanical stiffness of passive radiator suspension (inverse of compliance C_{mr})	$K_{ m mr}$	N/mm
Mechanical resistance of passive radiator suspension losses	$R_{ m mr}$	kg/s
Derived parameters:		
Q-factor of acoustic system at fb considering leakage losses	Q_1	

Resonance frequency of enclosure-port system	$f_{ m b}$	Hz	
Q-factor considering port losses	ort losses Q _p -		
Resonance frequency of enclosure-port system	$f_{ m f}$		
4.1.6 Room and Radiation			
4.1.6 Room and Radiation Radiation into half and full space: 2π or 4π (anechoic, piston)			
	Symbol	Unit	

4.2 Results

4.2.1 Electro-dynamical Transducer

Parameter	Symbol	Unit	
Derived parameters:			
Transducer resonance frequency (influences R_{ms} and M_{ms})	$f_{ m s}$	Hz	
Mechanical Q-factor of driver in free air, considering R_{ms} only	$Q_{ m ms}$		
Electrical Q-factor of driver in free air, considering Re only	$Q_{ m es}$		
Mechanical Q-factor of driver in free air, considering Rms only (influences R_{ms})	Q_{ts}		
Equivalent air volume of driver suspension	$V_{\rm as}$	1	
Efficiency and Sensitivity:			
Passband efficiency of driver operated in baffle	$\eta_{ ext{Pb}}$	%	
Passband sensitivity of driver operated in baffle with reference voltage u_{ref} and reference distance r_{ref} defined in ppg.	$L_{ m Pb}$	dB	

4.2.2 Enclosure

Parameter	Symbol	Unit		
Lumped parameters:				
Acoustical compliance of air in enclosure	C_{ab}	m³/Pa		
Mechanical stiffness of air in enclosure	$K_{ m mb}$	N/mm		
Acoustical compliance of air in front enclosure	\mathcal{C}_{f}	m³/Pa		
Total acoustical compliance of transducer and enclosure	\mathcal{C}_{at}	m³/Pa		
Total mechanical stiffness of transducer and enclosure	$K_{ m mt}$	N/mm		
System compliance ratio	α			
Derived parameters:				
Resonance frequency of the closed box system	$f_{\rm c}$	Hz		
Passive-Radiator resonance frequency (free air)	$f_{ m p}$	Hz		
Mechanical Q-factor of passive radiator in free air, considering R _{mr} only	$Q_{ m mp}$			
Total Q-factor considering all acoustical losses	$Q_{ m b}$			
Q-factor of the closed box system (considering system load)	$Q_{ m tc}$			

4.2.3 State Variables and Further Characteristics (depending on stimulus)

Parameter	Symbol	Unit
Reference Voltage-Sensitivity of selected stimulus for $r_{\rm ref}=1~{\rm m}$ and $u_{\rm ref}=1~{\rm V}$ according to IEC 60268-22	$L_{ m R}$	dB
Reference efficiency for selected stimulus according to IEC 60268-22	$\eta_{ m R}$	%
Far field SPL at distance r _{ref} for stimulus	$L_{ m p_{far}}$	dB
Terminal voltage (rms) for stimulus	$U_{\mathrm{T}_{rms}}$	V
Generator voltage (rms) for stimulus	$U_{G_{rms}}$	V
Terminal voltage (peak) for stimulus	$U_{\mathrm{T}_{\mathrm{peak}}}$	V
Generator voltage (peak) for stimulus	$U_{G_{\mathrm{peak}}}$	V
Input current (rms) for stimulus	$I_{\mathrm{T_{rms}}}$	A
Input current (peak) for stimulus	$I_{\mathrm{T}_{\mathrm{peak}}}$	A
Voice coil displacement (rms) for stimulus	$X_{c_{rms}}$	mm

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Voice coil displacement (peak) for stimulus	$X_{c_{ m peak}}$	mm
Voice coil velocity (rms) for stimulus	$V_{\rm c_{rms}}$	m/s
SPL in rear air volume for stimulus	$p_{ m box}$	dB
4.2.4 Transfer functions	1 501	
Function	Symbol	Unit
Voltage Sensitivity	L(f)	dB
Efficiency	$\eta(f)$	%
Electrical Impedance:	10)	,,,
Total electrical impedance	$Z_{\mathrm{e}}(f)$	Ω
Back EMF	$Blv/u_{\rm g}$	Ω
DC-Resistance of the transducer and the amplifier output resistance	$R_{\rm e} + R_{\rm g}$	Ω
Voice coil impedance	$Z_{\rm el}(f)$	Ω
Far Field Sound Pressure:	CI O y	
Total Sound Pressure	$H_{\mathrm{p_{far}}}(f,r)$	dB
Contribution from port	$H_{\mathfrak{p}}(f,r)$	dB
Target sound pressure	$H_{t}(f,r)$	dB
Total active system (with equalization)	$H_{\text{total}}(f,r)$	dB
Displacement divided by generator voltage:	total 0 · /	
Voice coil	$x_{\rm c}(f)/u_{\rm g}$	dB
Passive radiator	$x_{\rm r}(f)/u_{\rm g}$	dB
Velocity divided by generator voltage:	6	
Voice coil	$v_{\rm c}(f)/u_{\rm g}$	dB
Passive radiator	$v_{\rm r/p}(f)/u_{\rm g}$	dB
Force divided by generator voltage:	1/20 // 8	
At the motor	$F_{\rm c}(f)/u_{\rm g}$	dB
At M_{ms}	$F_{\rm Mms}(f)/u_{\rm g}$	dB
At R_{ms}	$F_{\rm Rms}(f)/u_{\rm g}$	dB
At C_{ms}	$F_{\rm Cms}(f)/u_{\rm g}$	dB
Into the acoustical system	$F_{\rm L}(f)/u_{\rm g}$	dB
Volume velocity divided by generator voltage:	- L () //g	
From S_d	$q_{\mathrm{S_d}}(f)/u_{\mathrm{g}}$	dB
Into \mathcal{C}_{ab}	$q_{\rm c}(f)/u_{\rm g}$	dB
Into C_f	$q_{\rm f}(f)/u_{\rm g}$	dB
Into R_{al}	$q_{\rm l}(f)/u_{\rm g}$	dB
Into port/passive radiator	$q_{\rm p}(f)/u_{\rm g}$	dB
Amplifier transfer function (voltage drop)	$u_{\rm t}(f)/u_{\rm g}$	dB
Prefilter transfer function (Equalizer)	$H_{\text{equ}}(f)$	dB
Stimulus Spectrum:	11equ ()	иБ
Relative input spectrum	$G_{\mathbf{w}}(f)$	dB
Aligned input spectrum	$G_{\rm eq}(f)$	dB
Voltage Spectrum:	Geq ()	иВ
Terminal voltage	u_{t}	dB
Amplifier output voltage without load	$u_{ m t}$	dB
Power Spectrum:	ug	ав
Electrical generator output power	$P_{ m e}$	dB
Acoustical output power	$P_{\rm a}$	dB
Power dissipation in amplifier	$P_{R_{\mathbf{g}}}$	dB
Spectrum of the sound pressure level		dB
State Spectrum:	$L_{ m p_{far}}$	ub
	Ī	dB
Voice Coil Volosity	$L_{x_{\text{coil}}}$	
Voice Coil Velocity	$L_{v_{ m coil}}$	dB
Voice Coil Force	$L_{F_{ m coil}}$	dB
Radiated Volume Velocity	L_{q_a}	dB

5 References S58

5 References

5.1	Related Modules	LPM Linear Parameter Measurement SIM Simulation SIM-AUR Simulation / Auralization
5.2	Manuals	LSIM Manual, as provided with dB-Lab 210.560 or higher
5.3	Related Papers	Wolfgang Klippel: "Green Speaker Design (Part 1: Optimal Use of System Resources)", 2019, Klippel GmbH
		Wolfgang Klippel: "Green Speaker Design (Part 2: Optimal Use of Transducer Resources)", 2019, Klippel GmbH
		R. H. Small: "Closed-Box Loudspeaker Systems", 2006, School of electrical Engineering, The University of Sydney, Australia

Find explanations for symbols at: http://www.klippel.de/know-how/literature.html

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